

Physically
consistent
simulation
of technical
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Michael Groß

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Physically consistent simulation of technical polymers

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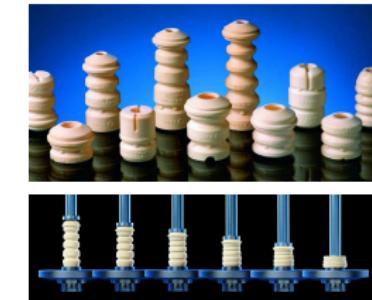
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Unfilled/Filled polymers

① Unfilled elastomers



Elastomer springs by **BASF Polyurethanes GmbH**

② Filled conductive polymers



Heat sinks and coil bodies by **CoolPoly**, $k=1,\dots,10 \text{ W/mK}$, $\rho > 1 \text{ T}\Omega \text{ cm}$, $\rho_0 = 1.47 \text{ g/cm}^3$

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Challenges of numerical simulation methods

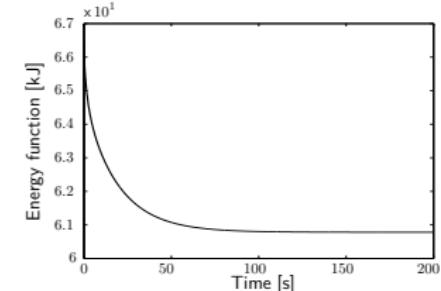
¹ Lion [1997], Reese & Govindjee [1998], Ehlers & Markert [2000], Miehe & Keck [2000], Haupt & Sedlan [2001], Rieger [2004], Kleuter [2007], Dal & Kaliske [2009], Méndez Diez [2010]² Simo et al. [1993], Ellsiepen [1999], Kirchner & Simeon [1999], Reese & Wriggers [1999], Doll et al. [2000], Hartmann [2003], Reese et. al [2005], Bischoff & Romero [2006]

① Realistic results

- ▶ Validated material laws¹
- ▶ Identified material param.¹
- ▶ Validated boundary cond.¹
- ▶ Locking-free space mesh²
- ▶ Momentum consistent &
- ▶ Energy consistent time mesh



Rotating heat pipe (McMaster University, Ca)



Energy balance relative to equilibrium

② Efficient methods

- ▶ Numerical stability
 - ~~ coarse space-time meshes
- ▶ Variable accuracy²
 - ~~ adaptive space-time meshing

Dynamics of solids

Oden [1972], Gurtin [1981], Miehe [1988], Holzapfel [2000], Andrews [2005], Chernyshev [2007], Fernández & Kuttler [2010]

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① Lagrangian local momentum balance

$$\boldsymbol{v}_t := \frac{\partial \boldsymbol{\varphi}_t}{\partial t}$$

$$\frac{\partial \boldsymbol{v}_t}{\partial t} = \frac{1}{\rho_0} \operatorname{DIV}[\boldsymbol{F}_t \boldsymbol{S}_t] + \boldsymbol{b}_t$$

② Deformation gradient/metric & strain measure

$$\boldsymbol{F}_t := \operatorname{GRAD}[\boldsymbol{\varphi}_t] \quad \boldsymbol{C}_t := [\boldsymbol{F}_t^b]^T \boldsymbol{F}_t \quad \boldsymbol{E}_t := \frac{1}{2} [\boldsymbol{C}_t - \boldsymbol{I}]$$

③ Piola/Mandel stress tensor

$$\boldsymbol{S}_t := 2 \frac{\partial \psi_t}{\partial \boldsymbol{C}} \quad \boldsymbol{\Sigma}_t^b := \boldsymbol{C}_t \boldsymbol{S}_t$$

④ Stress power & deformation rate tensor

$$\mathcal{P}_t := \boldsymbol{\Sigma}_t^b : \boldsymbol{L}_t^\sharp$$

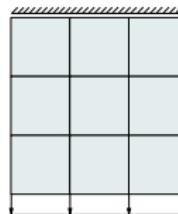
$$\boldsymbol{L}_t^\sharp := \frac{1}{2} \boldsymbol{C}_t^{-1} \frac{\partial \boldsymbol{C}}{\partial t}$$

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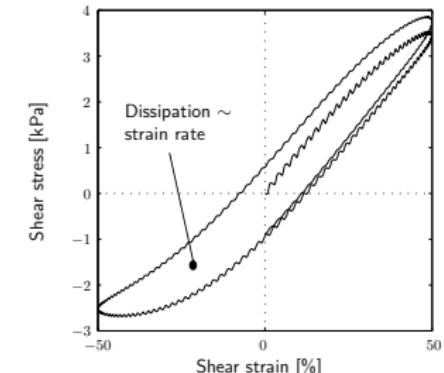
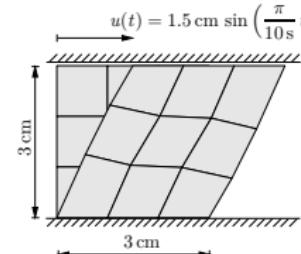
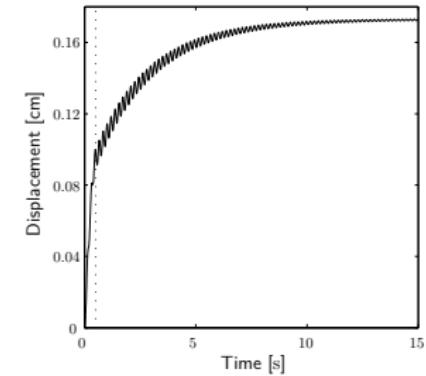
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Behaviour of viscoelastic elastomers

Miehe [1988], van den Bogert & de Borst [1994], Holzapfel & Simo [1996], Holzapfel [1996], Lion [1997], Reese & Govindjee [1998], Haupt [2002], Takács [2003]



$$-F_y(t) = \begin{cases} 4 \frac{\text{kN}}{\text{cm}} \sin\left(\frac{\pi}{\text{s}} t\right) & \text{für } 0 \text{ s} \leq t \leq 0.5 \text{ s} \\ 4 \frac{\text{kN}}{\text{cm}} & \text{für } 0.5 \text{ s} \leq t \leq 15 \text{ s} \end{cases}$$



$$S_t = f(E_t) + \Im_{\substack{0 \leq \sigma \leq s_\tau \\ 0 \leq \tau \leq t}} [\hat{E}_\sigma \circ s_\tau]$$

$$s_\tau = \int_0^\tau \|\dot{E}(t)\| dt \quad (\text{curve parameter of hysteresis})$$

Viscous evolution of isotropic elastomers

Miehe [1988], Le Tallec et al. [1993], Kaliske [1995], Reese & Govindjee [1998], Reese [2001], Kleuter [2007], Hartmann & Hamkar [2010]

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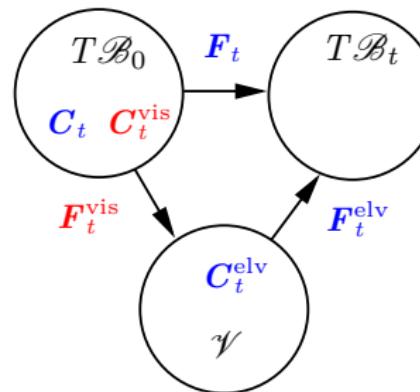
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1 Elastic metric tensor

$$\mathbf{C}_t^{\text{elv}} := [\mathbf{F}_t^{\text{vis}}]^{-T} \mathbf{C}_t [\mathbf{F}_t^{\text{vis}}]^{-1}$$

2 Elastic invariants

$$\mathfrak{I}(\mathbf{C}_t^{\text{elv}}) = \mathfrak{I}(\mathbf{C}_t [\mathbf{C}_t^{\text{vis}}]^{-1})$$

3 Viscous internal variable

$$\mathbf{C}_t^{\text{vis}} := [\mathbf{F}_t^{\text{vis},b}]^T \mathbf{F}_t^{\text{vis}}$$

4 Viscous evolution equation

$$\boldsymbol{\Sigma}_t^b = n_{\text{dim}} \mathbf{V}_{\text{sph}} \text{SPH}([\mathbf{L}_t^{\text{vis},\sharp}]^T) + 2 \mathbf{V}_{\text{dev}} \text{DEV}([\mathbf{L}_t^{\text{vis},\sharp}]^T)$$

5 Viscous Dissipation/rate tensor

$$D_t^{\text{vis}} := \boldsymbol{\Sigma}_t^b : \mathbf{L}_t^{\text{vis},\sharp} \geq 0$$

$$\mathbf{L}_t^{\text{vis},\sharp} := \frac{1}{2} [\mathbf{C}_t^{\text{vis}}]^{-1} \frac{\partial \mathbf{C}_t^{\text{vis}}}{\partial t}$$

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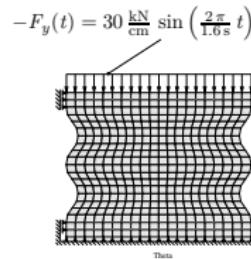
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	<i>H</i>	10 cm
	<i>W</i>	12.82 cm
Viscoelastic Simo-Taylor material		
Mass density	ρ_0	10 kg/cm ²
First Lamé constant	μ	7.5 kJ/cm ²
Second Lamé constant	λ	30 kJ/cm ²
Deviatoric viscosity	V_{dev}	10 kJ/s/cm ²
Spherical viscosity	V_{sph}	50 kJ/s/cm ²
Time step size	h_n	10 ms
Global iteration tolerance	tol	10^{-6} J
Local iteration tolerance	$tolovo$	10^{-9} J/cm ²
Number of spatial elements	n_{el}	400
Number of spatial nodes	n_{no}	441

Midpoint rule ($h_n = 10$ ms, $T = 1.15$ s)

eG method ($h_n = 10$ ms, $T = 5$ s)

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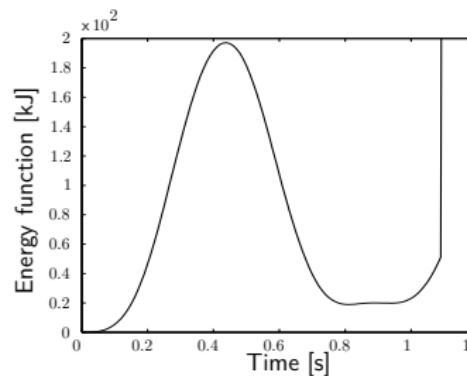
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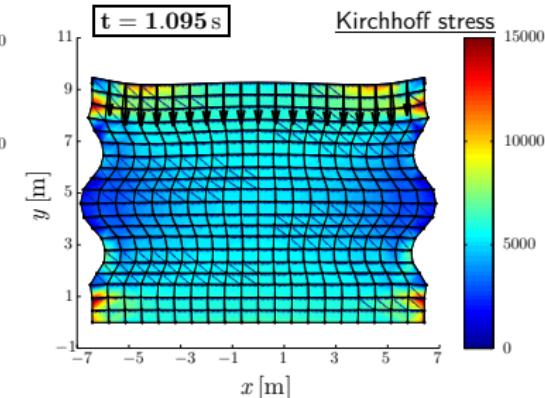
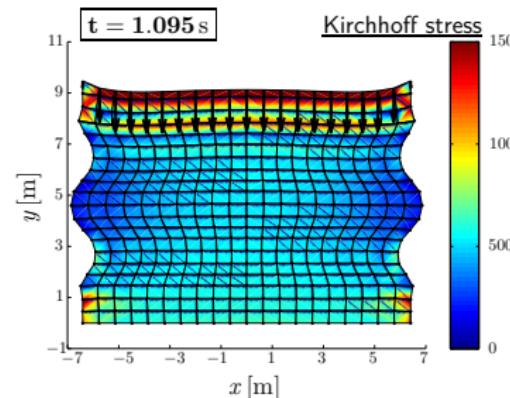
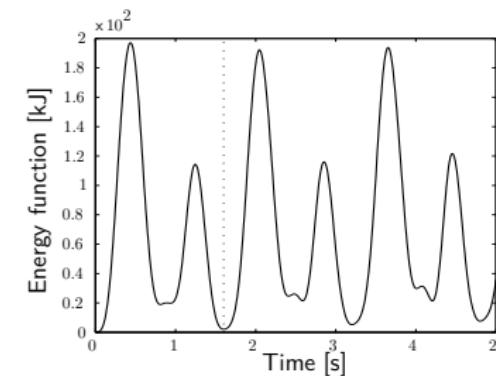
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Energy-Time behaviour & Stress distribution

Midpoint rule



eG method



Thermodynamics of elastomers

Oden [1972], Miehe [1988], Holzapfel & Simo [1996], Bérard et al. [1996], Lion [1997], Reese [2000], Boukamel et al. [2001]

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① Lagrangian local entropy balance

$$\eta_t := -\frac{\partial \psi_t}{\partial \theta}$$

$$\boxed{\frac{\partial \eta_t}{\partial t} = -\frac{1}{\theta_t} \operatorname{DIV}[\mathbf{Q}_t] + \frac{D_t^{\text{vis}}}{\theta_t} + \frac{r_t}{\theta_t}}$$

② Fourierian isotropic heat flux

$$\mathbf{q}_t := -\mathbf{k}_0 (J_t \mathbf{F}_t^\sharp)^{-T} \operatorname{GRAD}[\theta_t] \quad \mathbf{Q}_t := J_t \mathbf{F}_t^{-1} \mathbf{q}_t$$

③ Specific heat capacity & thermal expansion

$$\theta_t \frac{\partial \eta_t}{\partial \theta} =: \mathbf{c} \geq 0$$

$$\frac{1}{n_{\text{dim}}} \frac{\partial \operatorname{DET}[\mathbf{F}_t]}{\partial \theta} =: \beta \geq 0$$

④ Thermal dissipation

$$D_t^{\text{cds}} := -\operatorname{GRAD}[\ln \theta_t] \cdot \mathbf{Q}_t \geq 0$$

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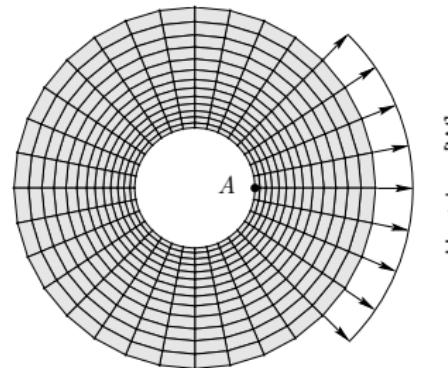
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Cooling of an elastomer spring (TE)



Heat loss [W]

Trapezoidal rule ($h_n = 10 \text{ ms}$, $T = 1.59 \text{ s}$)

Inner radius	R_i	0.5 m
Outer radius	R_o	1.5 m
Thermoelastic Simo-Taylor material		
Mass density	ρ_0	10 kg/m ²
First Lamé constant	μ	7.5 kJ/m ²
Second Lamé constant	λ	30 kJ/m ²
Specific heat capacity	c	0.3 kJ/m ² K
Heat expansion coefficient	β	10^{-4} K^{-1}
Heat conduction coefficient	k_0	0.3 kW/K
Ambient temperature	Θ_∞	298.15 K
Global iteration tolerance	tol	10^{-6} J
Number of spatial elements	n_{el}	416
Number of spatial nodes	n_{no}	448

ehG method ($h_n = 25 \text{ ms}$, $T = 15 \text{ s}$)

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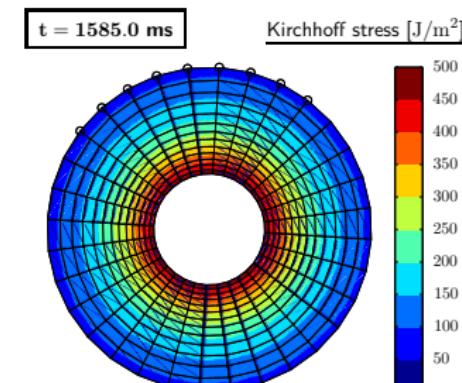
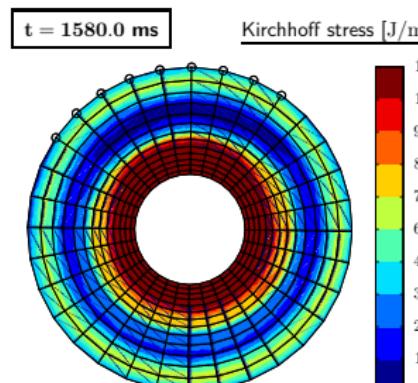
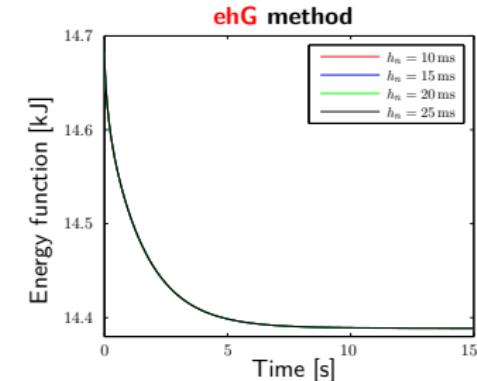
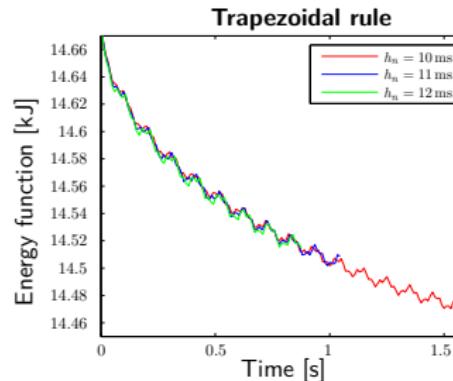
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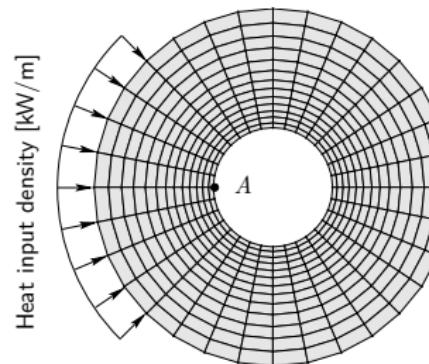
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Heating of an elastomer spring (TE)



Trapezoidal rule ($h_n = 10 \text{ ms}$, $T = 1.71 \text{ s}$)

Inner radius	R_i	0.5 m
Outer radius	R_a	1.5 m
Thermoelastic Simo-Taylor material		
Mass density	ρ_0	10 kg/m ²
First Lamé constant	μ	7.5 kJ/m ²
Second Lamé constant	λ	30 kJ/m ²
Specific heat capacity	c	0.3 kJ/m ² K
Heat expansion coefficient	β	10^{-4} K^{-1}
Heat conduction coefficient	k_0	0.3 kW/K
Ambient temperature	Θ_∞	298.15 K
Global iteration tolerance	tol	10^{-6} J
Number of spatial elements	n_{el}	416
Number of spatial nodes	n_{no}	448

ehG method ($h_n = 25 \text{ ms}$, $T = 10 \text{ s}$)

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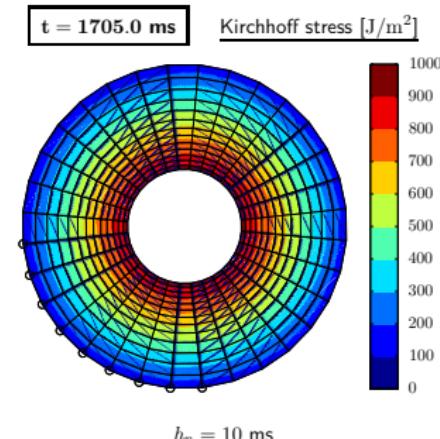
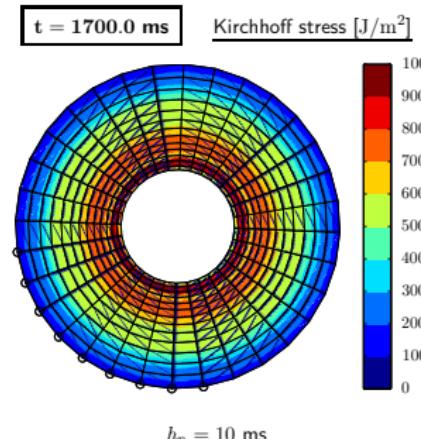
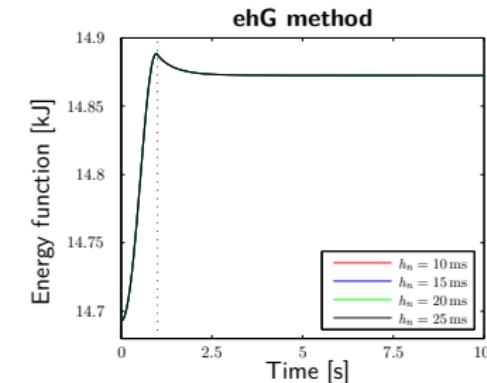
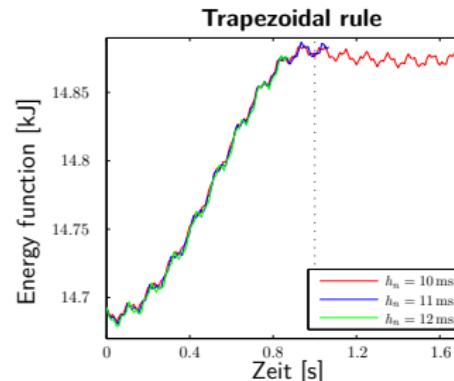
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Energy-consistent finite element method

Argyris & Scharpf [1969], Hughes et al. [1978], Betsch & Steinmann [2001], Hansbo [2001], Cockburn [2003], Bui [2007]

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1 Relative total energy

$$\mathcal{V}(t) = \int_{\mathcal{B}_0} \rho_0 \mathbf{v}_t \cdot \mathbf{v}_t + \psi_t + (\theta_t - \theta_\infty) \eta_t$$

2 Energy-consistent finite element method

$$\begin{aligned} \mathcal{V}(T) - \mathcal{V}(t_0) &= \int_{\mathcal{T}} \int_{\mathcal{B}_0} \rho_0 \mathbf{v}_t \cdot \frac{\partial \mathbf{v}_t}{\partial t} + \int_{\mathcal{T}} \int_{\mathcal{B}_0} \rho_0 \frac{\partial \mathbf{v}_t}{\partial t} \cdot \left[\frac{\partial \varphi_t}{\partial t} - \mathbf{v}_t \right] \\ &\quad + \int_{\mathcal{T}} \int_{\mathcal{B}_0} \frac{\partial \psi_t}{\partial \mathbf{F}} : \frac{\partial \mathbf{F}_t}{\partial t} - \int_{\mathcal{T}} \int_{\mathcal{B}_0} \frac{\partial \varphi_t}{\partial t} \cdot \left[\rho_0 \frac{\partial \mathbf{v}_t}{\partial t} - \text{DIV}[\mathbf{F}_t \mathbf{S}_t] \right] \\ &\quad + \int_{\mathcal{T}} \int_{\mathcal{B}_0} \frac{\partial \psi_t}{\partial \mathbf{C}^{\text{vis}}} : \frac{\partial \mathbf{C}_t^{\text{vis}}}{\partial t} + \int_{\mathcal{T}} \int_{\mathcal{B}_0} \frac{\partial \mathbf{C}^{\text{vis}}}{\partial t} : \left[\frac{1}{2} [\mathbf{C}_t^{\text{vis}}]^{-1} \boldsymbol{\Sigma}_t - \mathbf{Y}_t \right] \\ &\quad + \int_{\mathcal{T}} \int_{\mathcal{B}_0} (\theta_t - \theta_\infty) \frac{\partial \eta_t}{\partial t} - \int_{\mathcal{T}} \int_{\mathcal{B}_0} (\theta_t - \theta_\infty) \left[\frac{\partial \eta_t}{\partial t} + \frac{1}{\theta_t} \text{DIV}[\mathbf{Q}_t] - \frac{D_t^{\text{vis}}}{\theta_t} \right] \\ &\quad + \int_{\mathcal{T}} \int_{\mathcal{B}_0} \frac{\partial \theta_t}{\partial t} \left[\frac{\partial \psi_t}{\partial \theta_t} + \eta_t \right] \\ &\quad - \underbrace{\int_{\mathcal{T}} \int_{\mathcal{B}_0} \frac{\partial \mathbf{F}_t}{\partial t} : \left[\frac{\partial \psi_t}{\partial \mathbf{F}_t} - \mathbf{F}_t \mathbf{S}_t \right]}_0 \\ &= - \int_{\mathcal{T}} \int_{\mathcal{B}_0} \frac{\theta_\infty}{\theta_t} D_t^{\text{tot}} \leq 0 \end{aligned}$$

Discrete energy-consistent approximations

Gonzalez [1996], Armero & Romero [2001], Betsch & Steinmann [2001], Wriggers [2001], Kübler et al. [2003], Sansour et al. [2004], Bargmann [2006], Mohr et al. [2008]

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1 Discrete Galerkin methods

- ▶ Standard continuous Galerkin method where the test function is a time derivative
- ▶ Non-standard discontinuous Galerkin method with a special jump term elsewhere.

2 Non-standard approximation of the deformation metric

3 Non-standard approximation of the Mandel stress tensor

$$\Sigma_{\alpha}^{\text{num}, \flat} := \overline{\Sigma}_{\alpha}^{\flat} + \overline{\overline{M}}_{\alpha}^{\flat} + \overline{\Sigma}_{\alpha}^{\text{dam}, \flat} + \overline{\overline{M}}_{\alpha}^{\text{dam}, \flat}$$

4 Non-standard approximation of the entropy

5 Viscous damping

$$\overline{\Sigma}_{\alpha}^{\text{dam}, \flat} := \frac{\overline{J}_{\alpha}}{h_n} \left\{ n_{\text{dim}} \textcolor{blue}{D}_{\text{sph}} \text{SPH}(\overline{\overline{L}}_{\alpha}^{\sharp})^T + 2 \textcolor{blue}{D}_{\text{dev}} \text{DEV}(\overline{\overline{L}}_{\alpha}^{\sharp})^T \right\}$$

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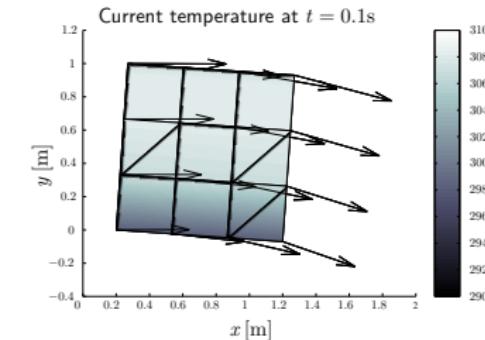
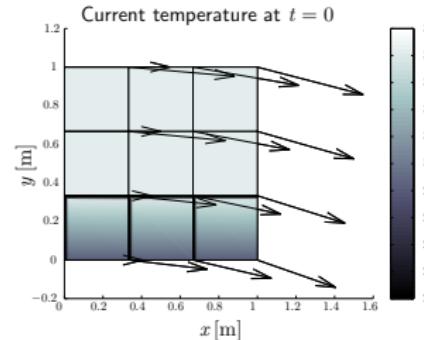
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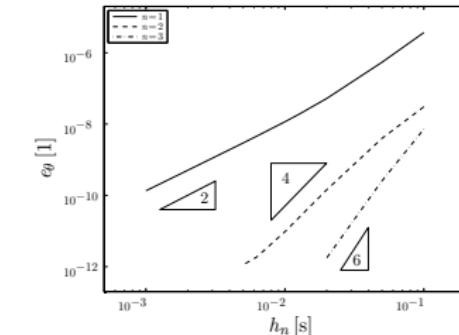
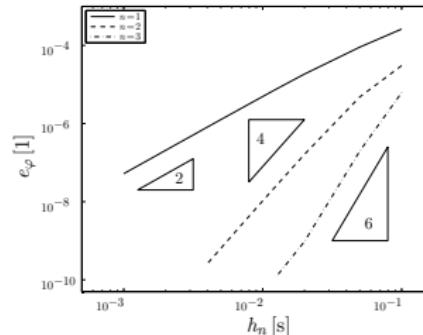
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Accuracy order of the energy-based integrator

① Free flying thermo-viscoelastic elastomer plate



② Relative error of position and temperature



Tension/Pressure of an elastomer spring (TVE)

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Temperature distribution of the ehG method ($h_n = 10 \text{ ms}$, $T = 5 \text{ s}$)

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Stress distribution of the ehG method ($h_n = 10 \text{ ms}$, $T = 5 \text{ s}$)

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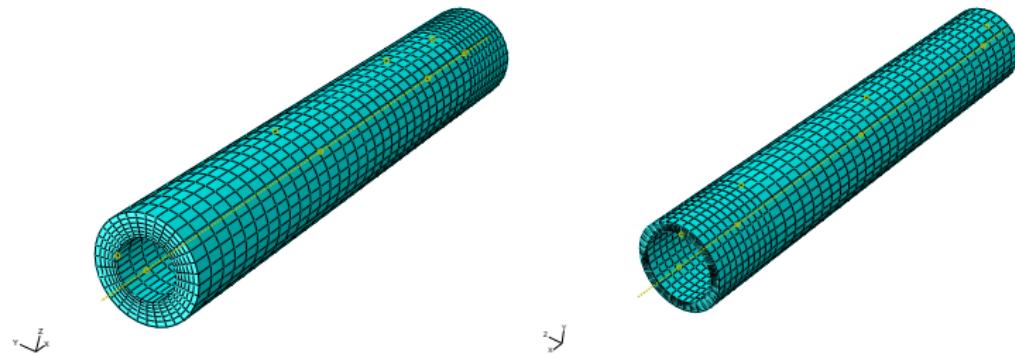
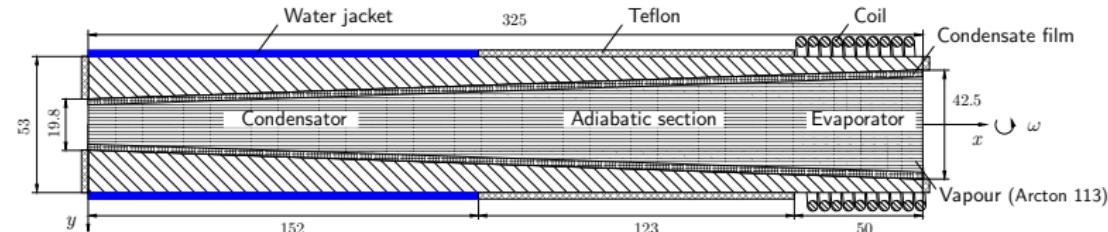
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Simulation of a rotating TVPE heat pipe

Daniels & Al-Jumaily [1974], Li et al. [1993], Harley & Faghri [1995], Streby & Ponnappan [1996], Song et al. [2003,2004,2008], Shukla et al. [2009]
Reich & Beer [1989], Reich et al. [1989], Weigand & Beer [1989,1992], Weigand [2004]



	Condensator	Adiabatic section	Evaporator
Outside	$\theta = \theta_\infty$ (Controlling)	$Q = 0$	Convection & Radiation
Inside	Convection & Pressure	Convection & Pressure	Convection & Pressure

Mechanical and thermal boundary conditions

Daniels & Al-Jumaily [1974], Harley & Faghri [1995], Baehr & Stephan [1996], Bay et al. [2003]

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① Inside pressure by gas expansion & radial acceleration

$$p = \rho_v \left\{ \frac{R \, T_v}{M_v} + \frac{\omega^2 r^2}{2} \right\}$$

② Convection on the condenser & adiabatic section

$$\overline{Q}_c = h_c A_c (T_s - \theta_c) \quad \overline{Q}_a = h_a A_a (T_s - \theta_a)$$

③ Convection on the evaporator

$$\overline{Q}_e = h_e A_e (\theta_e - T_s) \quad \overline{Q}_v = \alpha_v (\theta_e - \theta_\infty)$$

④ Heat radiation/input of the evaporator

$$\overline{Q}_v = \epsilon_e \sigma (\theta^4 - \theta_\infty^4) \quad r_e = \frac{Q_{inp}}{V_e}$$

⑤ Experimental data from the literature

Vapour (Ideal gas)	$\rho_v = 2.605 \text{ kg/m}^3$	$T_v = 346 \text{ K}$	$M_v = 0.18738 \text{ kg/mol}$
Condensate film	$T_s = 106^\circ \text{C}$		
	Condensator	Adiabatic section	Evaporator
Inside	$h_c = 483 \text{ W/m}^2 \text{K}$	$h_a = h_c$	$h_e = 4438 \text{ W/m}^2 \text{K}$
Outside			$\alpha_v = 13.566 \text{ W/m}^2 \text{K}$ $\epsilon_e = 0.92$

Plastic evolution equation of filled polymers

James & Green [1975], Bigg [1986], Heinle & Drummer [2010], Egelkraut et al. [2008]

Lion [1996], Miehe & Keck [2000], Haupt & Sedlan [2001], Haupt [2002], Hartmann & Neff [2003], Hartmann [2010]

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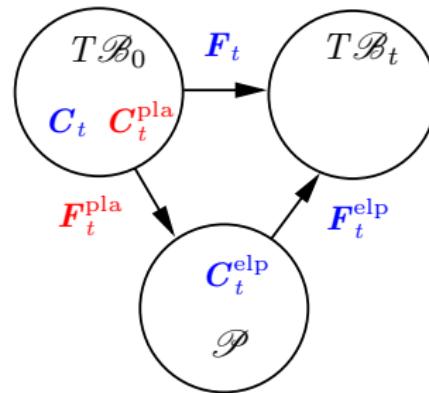
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➊ Plastic internal variable

$$\mathbf{C}_t^{\text{pla}} := [\mathbf{F}_t^{\text{pla},\flat}]^T \mathbf{F}_t^{\text{pla}}$$

➋ Plastic dissipation

$$\mathbf{D}_t^{\text{pla}} := \boldsymbol{\Sigma}_t^\flat : \mathbf{L}_t^{\text{pla},\sharp} \geqslant 0$$

➌ Plastic rate tensor

$$\mathbf{L}_t^{\text{pla},\sharp} := \frac{1}{2} [\mathbf{C}_t^{\text{pla}}]^{-1} \frac{\partial \mathbf{C}_t^{\text{pla}}}{\partial t}$$

➍ Plastic evolution equation

$$\boldsymbol{\Sigma}_t^\flat = n_{\text{dim}} V_{\text{sph}} \text{SPH}([\hat{\mathbf{L}}_t^{\text{pla},\sharp}]^T) + 2 V_{\text{dev}} \text{DEV}([\hat{\mathbf{L}}_t^{\text{pla},\sharp}]^T)$$

➎ Plastic evolution tensor

$$\hat{\mathbf{L}}_t^{\text{pla},\sharp} := \frac{1}{2} [\mathbf{C}_t^{\text{pla}}]^{-1} \frac{\partial \mathbf{C}_t^{\text{pla}}}{\partial \mathbf{s}} = \frac{1}{\|\mathbf{E}\|} \mathbf{L}_t^{\text{pla},\sharp}$$

Condensator section of the heat pipe

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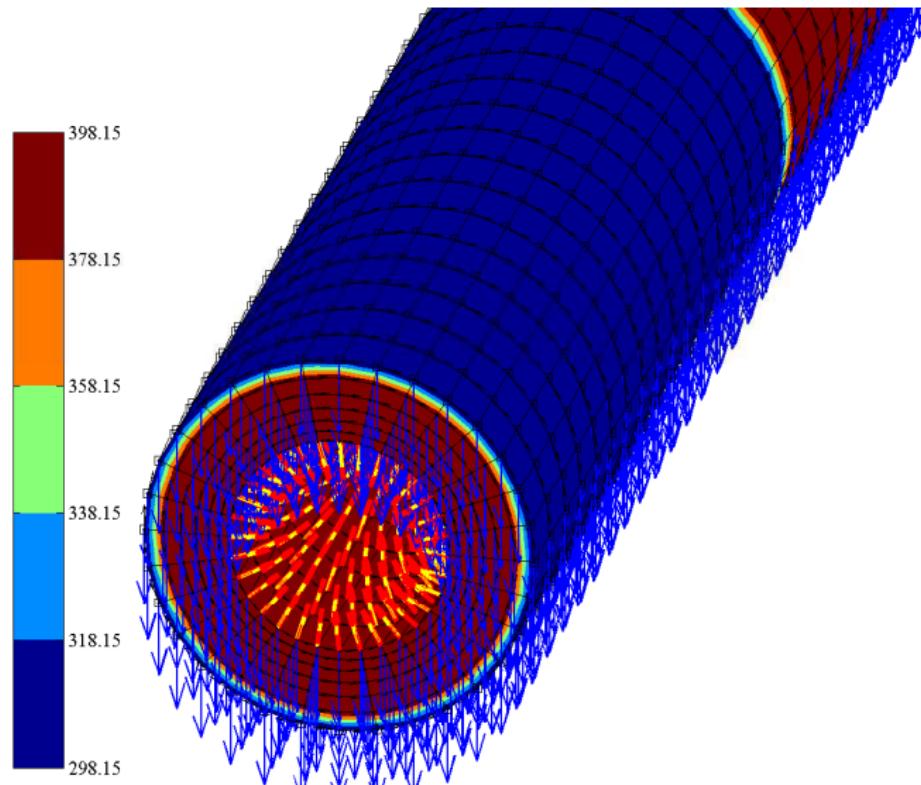
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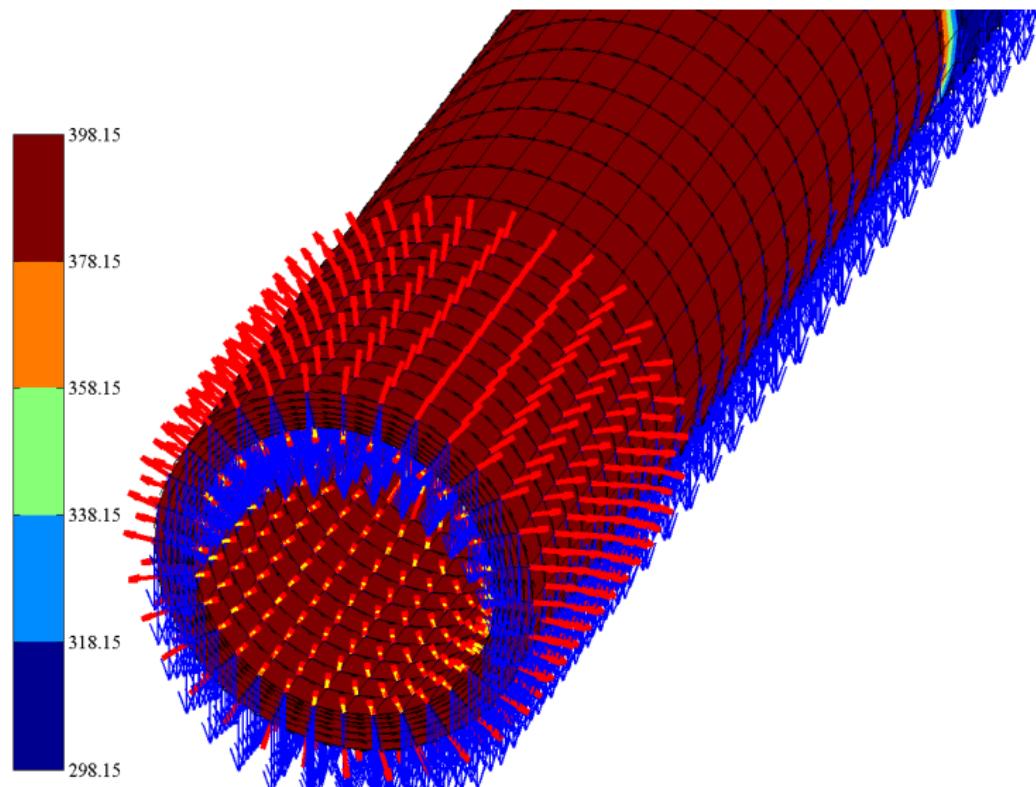
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Evaporator section of the heat pipe



Closing remarks: multiscale in time

Hughes & Stewart [1996], Bottasso[2002], Ladevèze & Nouy [2003], Kassiotis, Colliat, Ibrahimbegovic & Matthies [2009], Takizawa & Tezduyar [2011]

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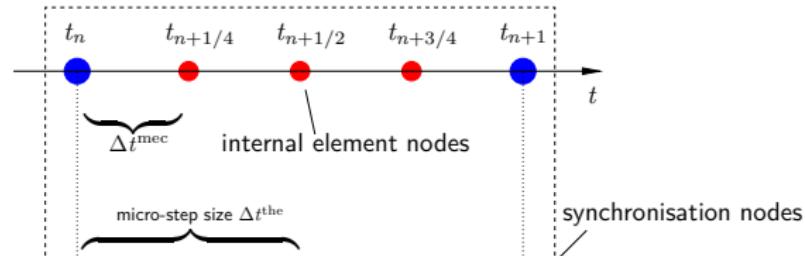
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① Advantages of the ehG method

- ▶ energy-momentum consistency
- ▶ higher-order accuracy
- ▶ **multiscale approximation** (1st step towards p -adaptivity)

② Thermo-mechanical multiscale approximation

mechanical residual (e.g. $k^{\text{mec}} = 4$)



thermal residual (e.g. $k^{\text{the}} = 2$)

macro-step size ΔT